

# Stability of Two-Wheeled Mobile Robot Using New Combined Pole-placement Method

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**Abstract:** In order to guarantee robotic stability, a new combined pole-placement method is proposed in this paper. The proposed method combines the linear quadratic regulator (LQR) into the pole-placement design. Firstly, a mathematical model of Two-Wheeled Mobile Robot (TWMR) is analytically derived from real TWMR. Secondly, the LQR for TWMR model is designed, and optimal poles can be obtained from the designed LQR. Thirdly, selection conditions of the best poles are the following; 1) the guarantee of convergence speed for TWMR system, 2) the number of vibration times for TWMR, 3) saturation evasion of the control input to TWMR's actuators, 4) the ratio of an imaginary part and a real part is carried out near one of selected poles. The pole-placement method selects the best poles than the optimal poles of LQR. Finally, the stability of the proposed method is confirmed by experiment results.

**Keywords:** inverted pendulum, linear quadratic regulator, pole placement method, two-wheeled mobile robot

## 1 INTRODUCTION

Two-wheeled mobile robot (TWMR) always exhibits many problems existing in industrial applications, for example, various nonlinear behaviors under different operation conditions, external disturbances, and physical constraints on some variables. Therefore, the task of on-line control of a highly nonlinear unstable has been a challenge for the modern control field. Since the system behaviors of TWMR including actuator dynamics are highly nonlinear, it is difficult to design a suitable control system that realizes real-time control and accurate balancing control at all time.

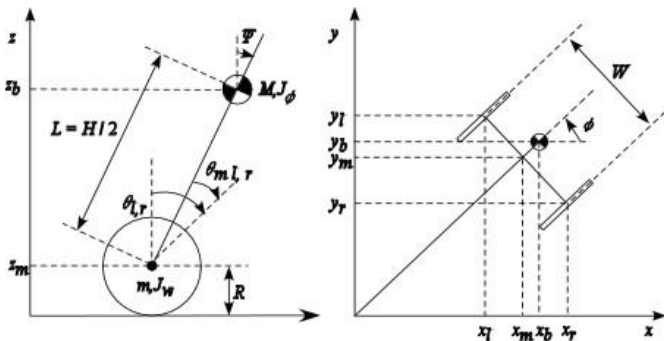
On the other hand, many research results of TWMR have been applied to various actual problems for designing walking gaits of humanoid robot, robotic wheelchairs, personal transport systems, and so on. Recently, the control problems of TWMR have been intensively studied due to the challenging demand of fast and precise performance. Besides protect-

ing safety of human and robotic operators, the robotic stability is very important research theme at the longest. There have been many studies on the stable problem, for example, sliding-model control [1], fuzzy switched swing-up control [2], and adaptive control [3] are researched on TWMR. But, these controls are very complex and the response time of the mobile robot system is slower.

## 2 TWMR'S MODEL

Figure 1 shows side view and plane view of TWMR, the coordinate system shown in Fig.1 is used TWMR model, and physical parameters of TWMR by measurements are shown in Table 1 [4].

We can drive motion equations from TWMR by the Lagrangian method based on the coordinate system shown in Fig.1, the generalized forces  $F_\theta$ ,  $F_\Psi$ ,  $F_\phi$  can be expressed



$\Psi$ : body pitch angle,  $\theta_{l,r}$ : left and right wheel angle,  
 $\theta_{m,l,r}$ : DC motor angle

Fig.1 Side and plane view of TWMR

Table 1 Physical parameters of TWMR

Gravity acceleration	$g=9.81$	(m/sec <sup>2</sup> )
Wheel weight	$m=0.03$	(kg)
Wheel radius	$R=0.04$	(m)
Wheel inertia moment	$J_w=mR^2/2$	(kgm <sup>2</sup> )
Body weight	$M=0.6$	(kg)
Body width	$W=0.14$	(m)
Body depth	$H=0.144$	(m)
Distance between mass and wheel axle	$L=H/2$	(m)
Body pitch inertia moment	$J_\Psi=ML^2/3$	(kgm <sup>2</sup> )
Body yaw inertia moment	$J_\phi=M(W^2+D^2)/12$	(kgm <sup>2</sup> )
DC motor inertia moment	$J_m=1 \times 10^{-5}$	(kgm <sup>2</sup> )
DC motor resistance	$R_m=6.69$	( $\Omega$ )
DC motor back EMF constant	$K_b=0.468$	(V sec/rad)
DC motor torque constant	$K_t=0.317$	(Nm/A)
Gear ratio	$n=1$	
Friction coef. of body and DC motor	$f_m=0.0022$	
Friction coef. between wheel and floor	$f_w=0$	

using the DC motor voltages  $v_l$  and  $v_r$ .

$$F_\theta = [(2m + M)R^2 + 2J_w + 2n^2 J_m] \ddot{\theta} + (MLR - 2n^2 J_m) \ddot{\psi} \quad (1)$$

$$F_\psi = (MLR - 2n^2 J_m) \ddot{\theta} + (ML^2 + J_\psi + 2n^2 J_m) \ddot{\psi} - MgL\psi \quad (2)$$

$$F_\phi = \left[ \frac{1}{2}mW^2 + \frac{W^2}{2R^2}(J_w + n^2 J_m) \right] \ddot{\phi} \quad (3)$$

here, consider the limit  $\Psi \rightarrow 0$ , and neglect the second order term like  $\Psi^2$ .

### 2.1 State space model

Now, from (1) to (3), we can obtain the following state space model of TWMR.

$$\dot{\mathbf{x}}_1(t) = \mathbf{A}_1 \mathbf{x}_1(t) + \mathbf{B}_1 \mathbf{u}(t) \quad (4)$$

$$\mathbf{y}_1(t) = \mathbf{C}_1 \mathbf{x}_1(t) \quad (5)$$

where, the state variable  $\mathbf{x}_1(t)=[\theta, \psi, \dot{\theta}, \dot{\psi}]^T$ , the input  $\mathbf{u}(t)=[v_r, v_l]^T$ , and the output  $\mathbf{y}_1(t)=[\theta, \psi, \dot{\theta}, \dot{\psi}]^T$ , the parameter matrices of the state space model (4), (5) describe the following.

$$\mathbf{A}_1 = \begin{bmatrix} 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 1 \\ 0 & A_1(3,2) & A_1(3,3) & A_1(3,4) \\ 0 & A_1(4,2) & A_1(4,3) & A_1(4,4) \end{bmatrix} \quad (6)$$

$$\mathbf{B}_1 = \begin{bmatrix} 0 & 0 \\ 0 & 0 \\ B_1(3,1) & B_1(3,2) \\ B_1(4,1) & B_1(4,2) \end{bmatrix} \quad (7)$$

$$\mathbf{C}_1 = \text{diag} [1 \ 1 \ 1 \ 1] \quad (8)$$

$$\mathbf{E} = \begin{bmatrix} E(1,1) & E(1,2) \\ E(2,1) & E(2,2) \end{bmatrix} \quad (9)$$

$$A_1(3,2) = -\frac{gMLE(1,2)}{\det \mathbf{E}}$$

$$A_1(4,3) = \frac{gMLE(1,1)}{\det \mathbf{E}}$$

$$A_1(3,3) = -\frac{2[(\beta + f_w)E(2,2) + \beta E(1,2)]}{\det \mathbf{E}}$$

$$A_1(3,4) = \frac{2[(\beta + f_w)E(1,2) + \beta E(1,1)]}{\det \mathbf{E}}$$

$$A_1(4,3) = \frac{2\beta [E(2,2) + E(1,2)]}{\det \mathbf{E}}$$

$$A_1(4,4) = -\frac{2\beta [E(1,1) + E(1,2)]}{\det \mathbf{E}}$$

$$B_1(3,1) = \frac{\alpha [E(2,2) + E(1,2)]}{\det \mathbf{E}}$$

$$B_1(4,1) = -\frac{\alpha [E(1,1) + E(1,2)]}{\det \mathbf{E}}$$

$$\det(\mathbf{E}) = E(1,1)E(2,2) - E(1,2)^2$$

$$E(1,1) = (2m + M)R^2 + 2J_w + 2n^2 J_m$$

$$E(1,2) = MLR - 2n^2 J_m$$

$$E(2,1) = MLR - 2n^2 J_m$$

$$E(2,2) = ML^2 + J_\psi + 2n^2 J_m$$

$$\alpha = \frac{nK_t}{R_m}, \quad \beta = \frac{nK_t K_b}{R_m} + f_m$$

### 2.2 LQR design for TWMR

In this section, LQR design for TWMR is considered as an expansive state space equation. We select  $\theta$  as a reference state  $\theta_{ref} = \mathbf{C}_\theta \mathbf{x}_{ref}$ , and define an error  $\mathbf{e}(t) = \mathbf{C}_\theta \mathbf{x}_{ref} - \theta(t)$ ,  $\dot{z}(t) = \mathbf{e}(t)$ , then the expansive state space equation can be expressed by

$$\dot{\bar{\mathbf{x}}}(t) = \bar{\mathbf{A}} \bar{\mathbf{x}}(t) + \bar{\mathbf{B}} \bar{\mathbf{u}}(t) \quad (10)$$

$$\bar{\mathbf{x}}(t) = \begin{bmatrix} \dot{\mathbf{x}}(t) \\ \mathbf{z}(t) \end{bmatrix}, \quad \bar{\mathbf{A}} = \begin{bmatrix} \mathbf{A}_1 & 0 \\ 0 & \mathbf{C}_\theta \end{bmatrix},$$

$$\bar{\mathbf{B}} = \begin{bmatrix} \mathbf{B}_1 & 0 \\ 0 & -\mathbf{C}_\theta \end{bmatrix}, \quad \bar{\mathbf{u}}(t) = \begin{bmatrix} \mathbf{u}(t) \\ \mathbf{x}_1(t) \end{bmatrix}$$

According to an optimal feedback gain matrix  $\mathbf{K}$  such that the feedback control law

$$\bar{\mathbf{u}}^*(t) = -\mathbf{K} \bar{\mathbf{x}}(t) \quad (11)$$

$$= \begin{bmatrix} -\mathbf{k}_f & -\mathbf{k}_i \end{bmatrix} \begin{bmatrix} \mathbf{x}(t) \\ \mathbf{z}(t) \end{bmatrix} \quad (12)$$

$$\mathbf{K} = \mathbf{R}^{-1} \bar{\mathbf{B}}^T \mathbf{P} \quad (13)$$

minimizes the performance index

$$J = \int_0^\infty (\bar{\mathbf{x}}^T(t) \mathbf{Q} \bar{\mathbf{x}}(t) + \bar{\mathbf{u}}^T(t) \mathbf{R} \bar{\mathbf{u}}(t)) dt \quad (14)$$

subject to the constraint equation (10). Here, matrix  $\mathbf{P}$  of (13) is the unique positive-definite solution of the associated matrix Riccati Equation

$$\bar{\mathbf{A}}^T \mathbf{P} + \mathbf{P} \bar{\mathbf{A}} + \mathbf{Q} - \mathbf{P} \bar{\mathbf{B}} \mathbf{R}^{-1} \bar{\mathbf{B}}^T \mathbf{P} = 0 \quad (15)$$

As stated above, a servo control for TWMR is shown in Fig.2, where  $\mathbf{C}_\theta = \text{diag} [1, 0, 0, 0]$ , and the weight matrices  $\mathbf{Q}$ ,  $\mathbf{R}$  of (13) by experimental trial and error.

$$\mathbf{Q} = \text{diag} [1 \ 6 \times 10^5 \ 1 \ 1 \ 4 \times 10^2] \quad (16)$$

$$\mathbf{R} = \text{diag} [1 \times 10^3 \ 1 \times 10^3] \quad (17)$$

To calculate the LQR problem, and obtain the gain  $\mathbf{K}$  is

$$\mathbf{K} = [k_\theta, k_\psi, k_{\dot{\theta}}, k_{\dot{\psi}}, k_i] \quad (18)$$

$$\mathbf{K} = [-0.867, -31.931, -1.154, -2.783, -0.446]$$

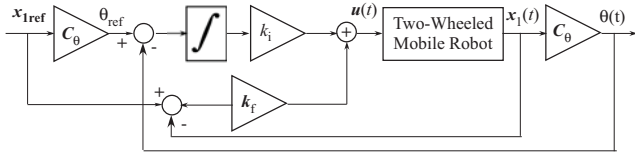


Fig. 2 Servo control system block diagram

### 3 PROPOSED METHOD

In this section, we present a design proposed method called a combined pole-placement method. We assume that all state variables are measurable and are available for feedback. If the system considered is completely state controllable, then poles of the closed-loop system may be placed at any desired locations by means of state feedback through an appropriate state feedback gain matrix.

The present design technique begins with a determination of the desired closed-loop poles based on the transient-response and frequency-response requirements, such as speed, damping ratio, and bandwidth, as well as steady-state requirements.

#### 3.1 Pole-placement for TWMR

From the expansive state space equation (10) and the optimal control input (11), the closed-loop system can be expressed as

$$\begin{aligned} \dot{\bar{x}}(t) &= \bar{A}\bar{x}(t) - \bar{B}K\bar{x}(t) \\ &= (\bar{A} - \bar{B}K)\bar{x}(t) \end{aligned} \quad (19)$$

The desired characteristic equation is

$$\begin{aligned} |sI - \bar{A} + \bar{B}K| &= (s - p_1)(s - p_2) \cdots (s - p_n) \\ &= s^n + a_{n-1}s^{n-1} + \cdots + a_0 = 0 \end{aligned} \quad (20)$$

where,  $p_i$  ( $i=1, 2, \dots, n$ ) are poles of the closed-loop system (19). The first step in the pole-placement design approach is to choose the locations of the desired closed-loop poles. The most frequently used approach is to choose such poles on

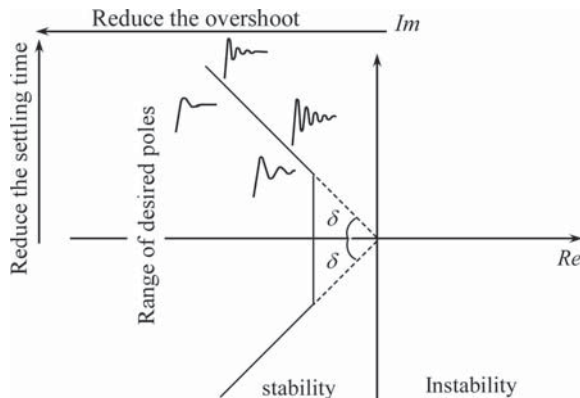


Fig.3 Range of desired poles

basis of experience in root-locus design, placing a dominant pair of closed-loop poles and choosing other poles so that are far to the left of the dominant closed-loop poles.

We have to note that if the dominant closed-loop poles is far from the  $j\omega$ -axis, so that the system response becomes very fast, the signals in the system become very large, with the result that system may become nonlinear, this should be avoided and shown in Fig.3.

Another approach is based on LQR approach and determines the desired closed-loop poles such that the system balances between the acceptable response and the amount of control energy required.

In the complex coordinate of Fig.3, an angle  $\delta = \tan^{-1}(\sqrt{1 - \zeta^2}/\zeta)$ , here  $\zeta$  is a damping ratio. For servo control system of robotics, TWMR and so on,  $\zeta = 0.6 - 0.8$  and  $\delta = 37^\circ - 53^\circ$  have been generally accepted [5].

#### 3.2 Choosing the location of desired closed-loop poles

From  $K$  of (18), the pole  $P_0$  can be derived and shows the following

$$P_0 = [-284.41, -10.92, -1.3 \pm j0.73, -1.47] \quad (21)$$

We change the damping ratio  $\zeta$  with 0.05 step from 0.6 to 0.8, and obtain poles  $P_i$  ( $i = 1, \dots, 5$ ) their feedback gains, the  $\zeta$  and the natural frequency  $\omega_n$  are shown in Table 2, 3 and 4, respectively.

Table 2 Closed-loop poles

	$p_1$	$p_2$	$p_3, p_4$	$p_5$
$P_0$	-284.41	-10.9	$-1.3 \pm j0.73$	-1.47
$P_1$	-284.41	-10.9	$-1.3 \pm j0.96$	-1.47
$P_2$	-284.41	-10.9	$-1.3 \pm j1.15$	-1.47
$P_3$	-284.41	-10.9	$-1.3 \pm j1.31$	-1.47
$P_4$	-284.41	-10.9	$-1.3 \pm j1.51$	-1.47
$P_5$	-284.41	-10.9	$-1.3 \pm j1.73$	-1.47

Table 3 Feedback gains

	$k_\theta$	$k_\psi$	$k_{\dot{\theta}}$	$k_{\dot{\psi}}$	$k_i$
$P_0$	-0.867	-31.931	-1.154	-2.783	-0.446
$P_1$	-0.927	-32.059	-1.160	-2.797	-0.524
$P_2$	-0.990	-32.191	-1.167	-2.810	-0.604
$P_3$	-1.051	-32.320	-1.173	-2.823	-0.683
$P_4$	-1.138	-32.506	-1.182	-2.841	-0.796
$P_5$	-1.249	-32.741	-1.193	-2.865	-0.939

Table 4 Damping ratio and Natural frequency

	Damping ratio $\zeta$	Natural frequency $\omega_n$ (rad/s)
$P_0$	0.87	1.49
$P_1$	0.80	1.62
$P_2$	0.75	1.74
$P_3$	0.70	1.85
$P_4$	0.65	1.99
$P_5$	0.60	2.16

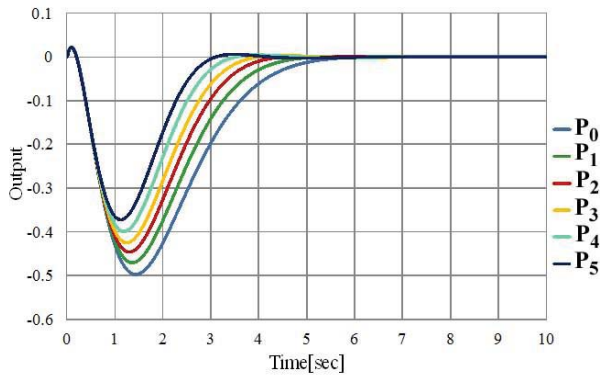


Fig.4 Unit step responses with  $P_i$

Figure 4 shows unit step responses of the closed-loop with  $P_i$ . On the same method, we keep the  $\zeta$ , and change the real number of the conjugate number  $-1.3 \pm j0.73$  with 0.2 step to negative direction. We obtain the good pole  $P_6 = [-284.4, -10.9, -1.5 \pm j0.84, -1.47]$  with gain margin 5.0 [dB], resonant peak -10.1 [dB], setting time 4.79 (sec.), overshoot -0.41, and  $\zeta = 0.87, \omega_n = 1.72$  (rad/sec.).

#### 4 EXPERIMENTAL RESULTS

A real TWMR is established practically in our laboratory to illustrate the effectiveness of the proposed method. Using the same parameters shown in Table 1, real stable position experiment is implemented to verify the feasibility of the proposed method with the pole  $P_6$  and the LQR method with  $P_0$ , respectively.

Fig.5 presents the body angle  $\Psi$  which tests the stability of TWMR, a small shaking of the body angle with the pole  $P_6$  than the pole  $P_0$ . Fig.6 shows the shifting distance of TWMR at an appointed position. A large distance error using the pole  $P_0$  is produced, but, the proposed method with the pole  $P_6$  generates very small movement from the appointed position.

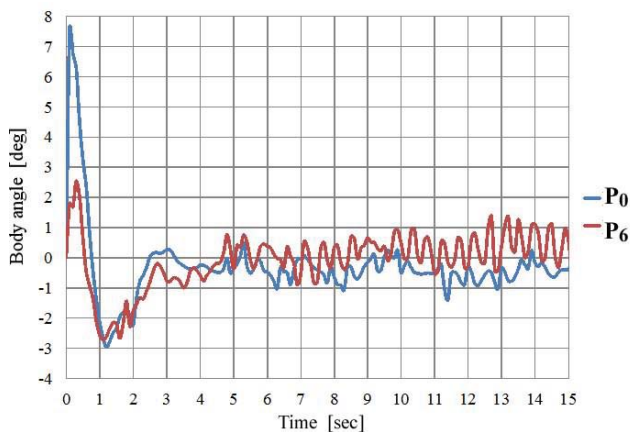


Fig.5 Experimental results of body angle

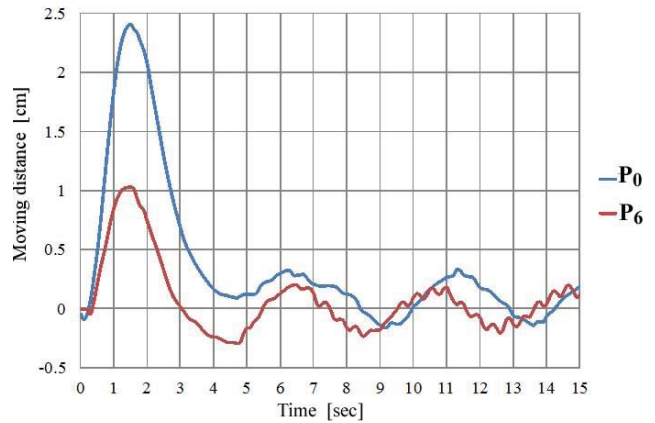


Fig.6 Experimental results of shifting distance

#### 5 CONCLUSION

This paper has implemented the LQR method and pole-placement developmental hardware and software on the real TWMR system, the combined pole-placement method has been newly proposed to control TWMR achieving the desired control behaviors than LQR method.

Finally, the experiment of stable position control was executed using the proposed method and LQR method, the experimental results indeed have verified that the proposed method is effective for the real TWMR system.

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